

# LATEST NEWS FROM STOCHASTIC COOLING DEVELOPMENTS FOR THE COLLECTOR RING AT FAIR

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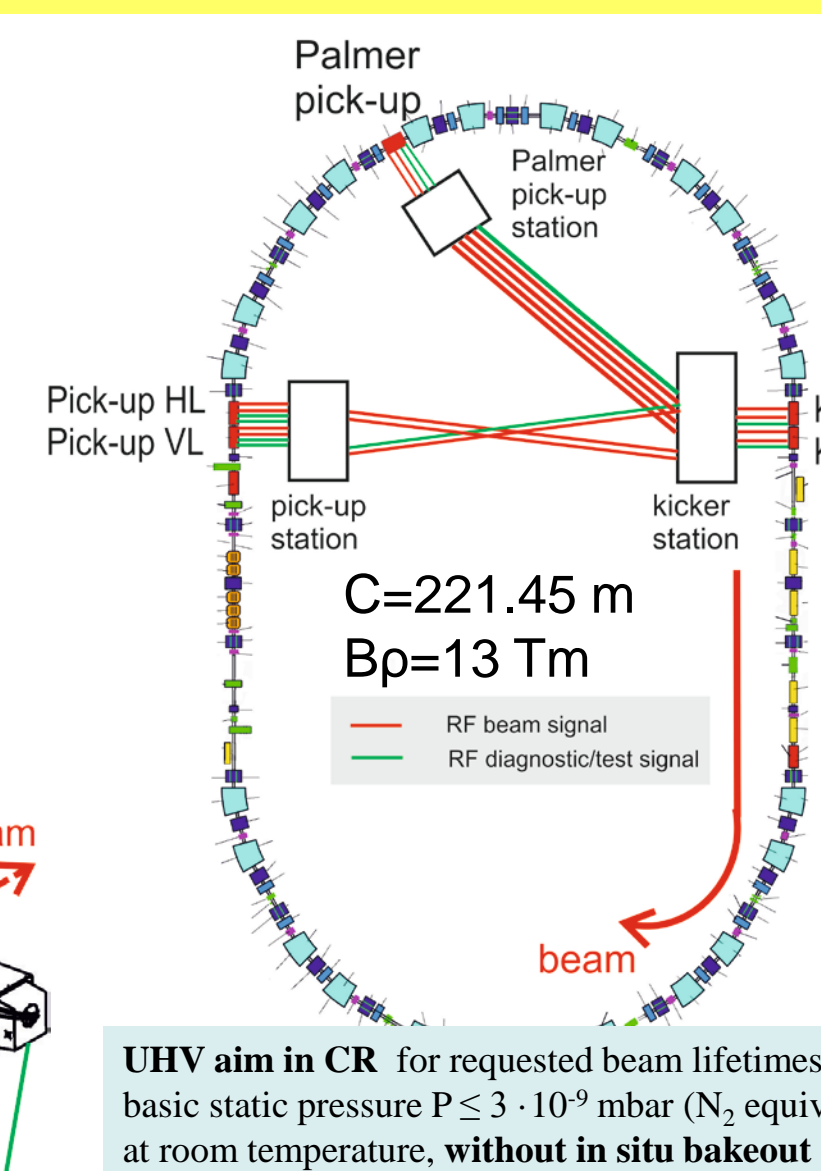
## CR stochastic cooling

for antiprotons  $v=0.97 c$  ( $f_{rev}=1.315$  MHz)  
for ions/RIBs  $v=0.83 c$  ( $f_{rev}=1.124$  MHz)

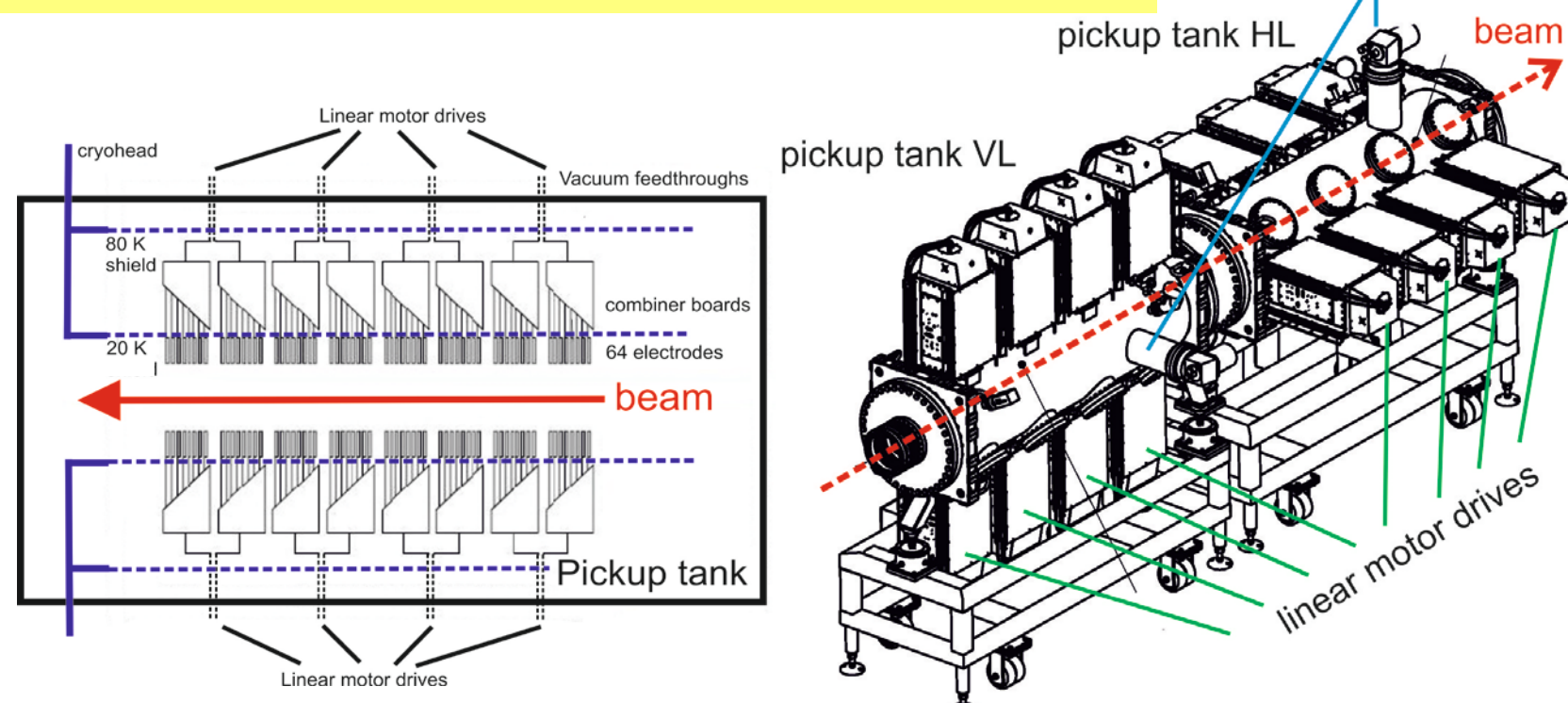
System bandwidth=1-2 GHz

Main issue for antiprotons: increase ratio  
Schottky signal ( $\propto Q^2$ )  
thermal noise

- Pick-up electrodes cooled at 20-30K
- Plunging pick-up electrodes i.e. synchronously moving closer to the beam during cooling
- Notch filter longitudinal cooling for noise suppression around revolution harmonics



UHV aim in CR for requested beam lifetimes of 100 s:  
basic static pressure  $P \leq 3 \cdot 10^{-9}$  mbar ( $N_2$  equivalent)  
at room temperature, without in situ bakeout



UHV, cryogenic and plunging challenges!

## Damping of unwanted microwave modes

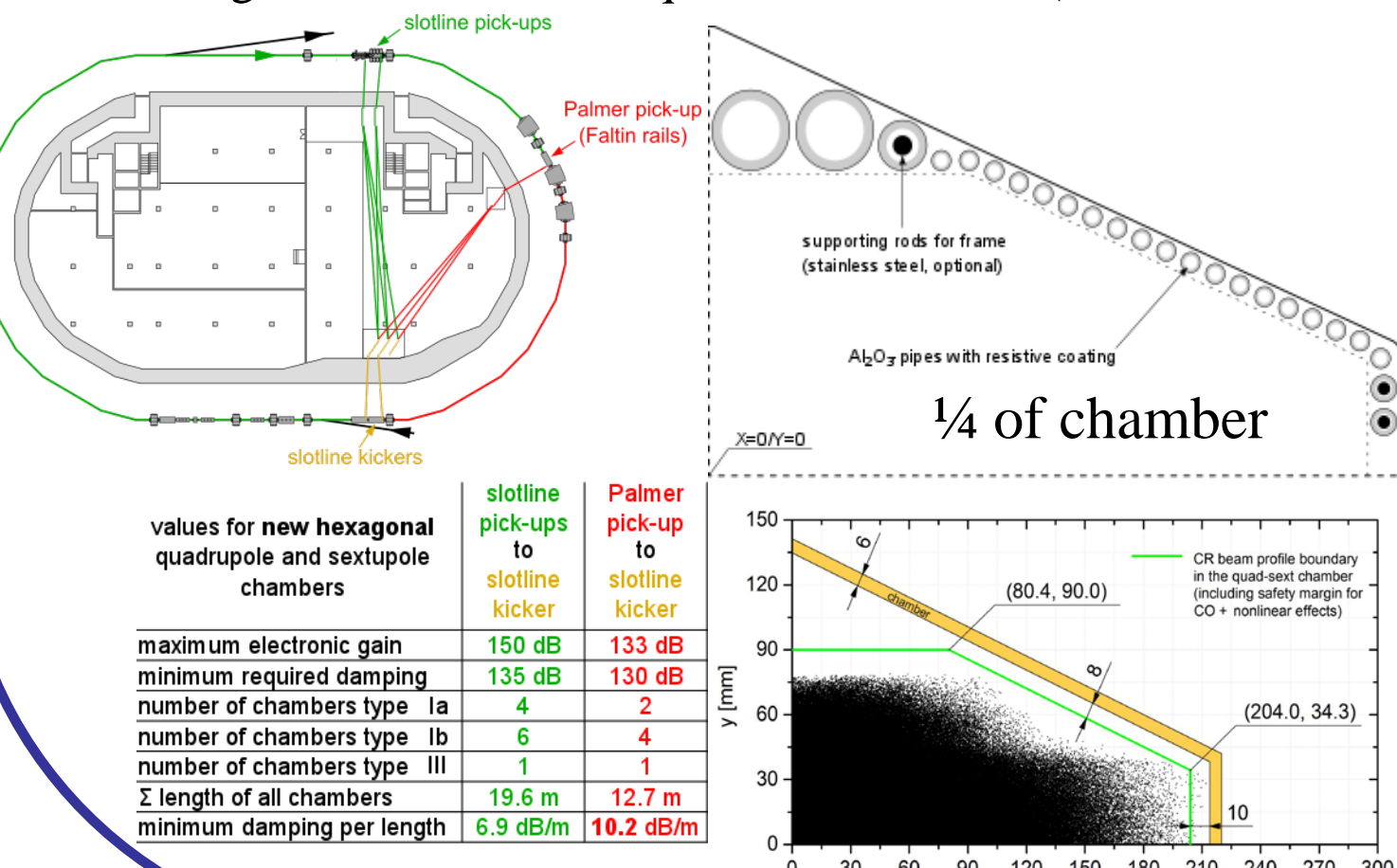
- Large-aperture PU/KI/magnet vacuum chambers in between: many propagating modes
- High gain (>130 dB) in signal paths for fast cooling: short beamline Palmer PU-KI
- high microwave damping requirements in the band 1-2 GHz
- Need UHV-compatible microwave absorbers, NO in situ bakeout in the CR!

inside magnet vacuum chambers, between PUs and KI in the arcs:  
 $Al_2O_3$  ceramic tubes with resistive outside coating ( $\leq 60 m^2$  per arc)

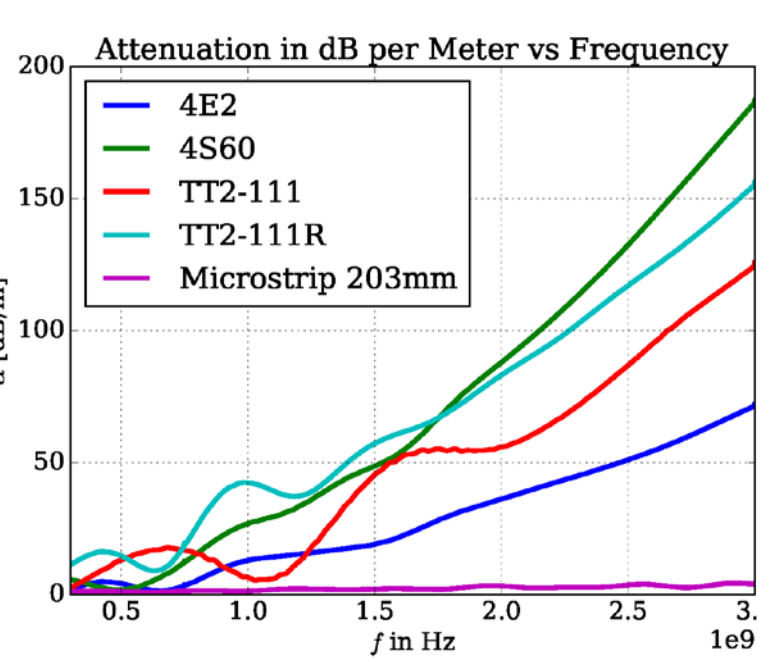
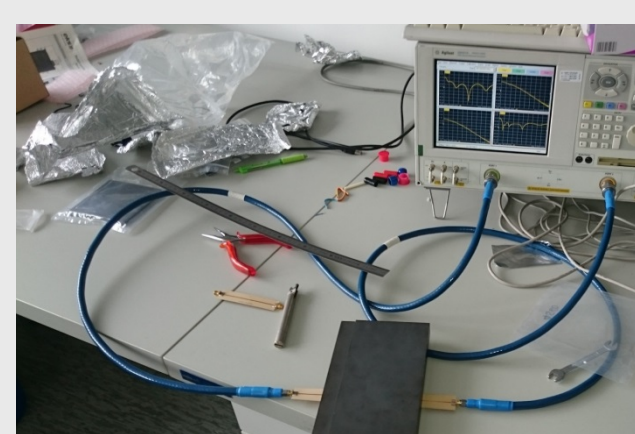
ESR quad chamber with damping tubes for mw modes

The ESR module is UHV-compatible to stainless steel ( $1.4 \cdot 10^{-11}$  mbar-l/s-cm<sup>2</sup> after 10 h of pumping without bakeout)

The concept: MW damping tube modules inside all hexagonal quadrupole/sextupole chambers in the CR arcs.  
4x24  $Al_2O_3$  tubes (4 diameters from  $\phi 6$  mm to  $\phi 24$  mm), with coating sheet resistance  $R_{sq} = 150 \Omega/\square \pm 30\%$  (HFSS simulation).

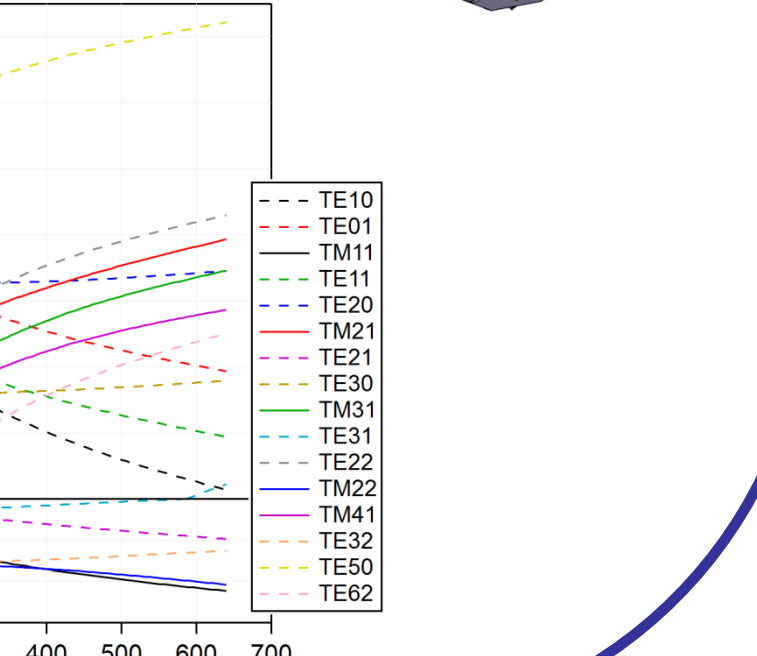
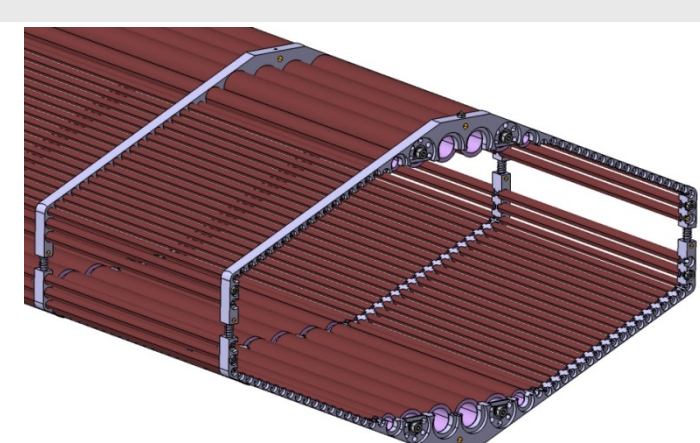


Inside PU/KI tanks: ferrites ( $\leq 2 m^2$  per tank)



Outgassing rate of TT2-111R and 4S60 UHV-compatible:  
x2-3 of stainless steel ( $< 1.6 \cdot 10^{-10}$  mbar-l/s-cm<sup>2</sup> after 1 week of pumping without bakeout)

Min. max values TT2-111R at room temperature from E. Chojnacki et al., (Cornell), DC Conductivity of RF absorbing materials, Proc. SRF2009, Berlin, Germany (2009).



## Simulations of antiproton cooling performance

Reference (most critical) case:  
10<sup>8</sup> antiprotons at 3 GeV  
cooling for 10 s in all 3 phase-space planes  
1-2 GHz band

conservative = no PU electrode plunging  
= assuming el. gain transverse=const.

Already with these assumptions cooling performance close to the design limits for the HESR downstream of the CR.

Longitudinal notch filter cooling (CERN code Fokker Planck)

best for constant el. gain=150 dB ( $3.2 \cdot 10^7$ )  
no safety margin, high power requirement

Transverse cooling with 'rms' model

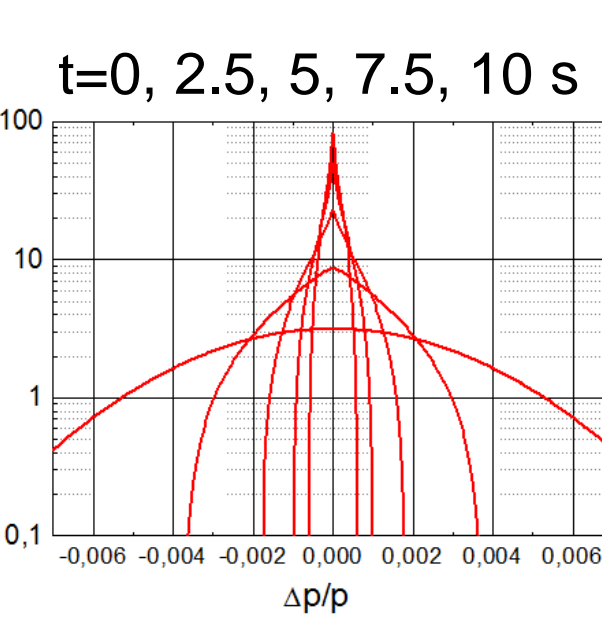
$$-\frac{1}{\varepsilon} \frac{d\varepsilon}{dt} = \frac{1}{\tau_{\perp}} = \frac{2W}{N} [2gB - g^2(M+U)]$$

Mixing KI-PU  
Interplay between transv. & long. cooling

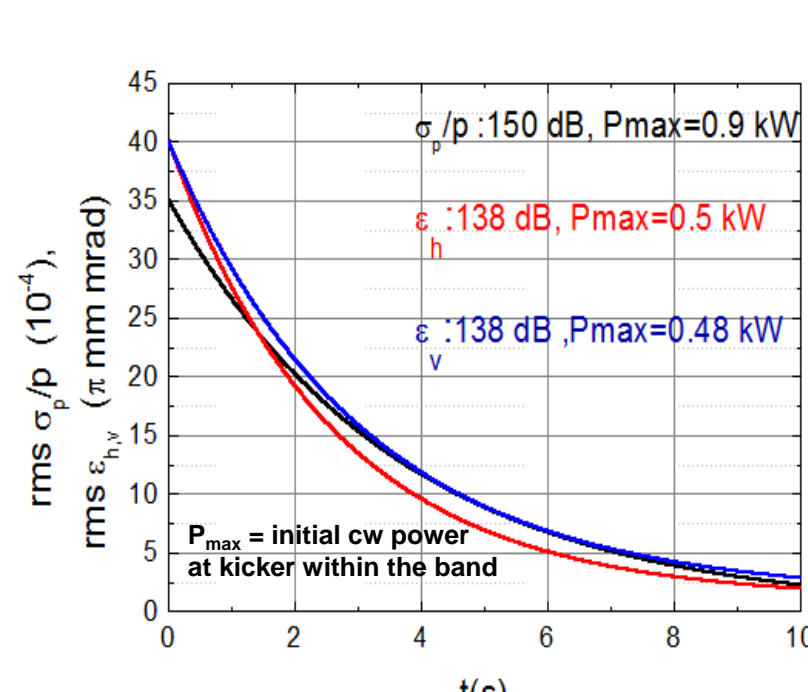
Simultaneous notch filter long. cooling ON.  
Ansatz from Fokker-Planck results at 150 dB

best for assumed const. el. gain=138 dB  
some margin @ decreasing gain and plunging

- pbar lattice:  $\eta=0.014$  ( $\eta_{\text{res}}=4.84$ ),  $\eta_{\text{pk}} \approx 0$  (suppresses undesired mixing).
- Response of designed slotline PU/KI included
- Feedback by the beam in long. and transverse model
- $T_{\text{eff}}$  at preamplifier 100 K (amp. noise)

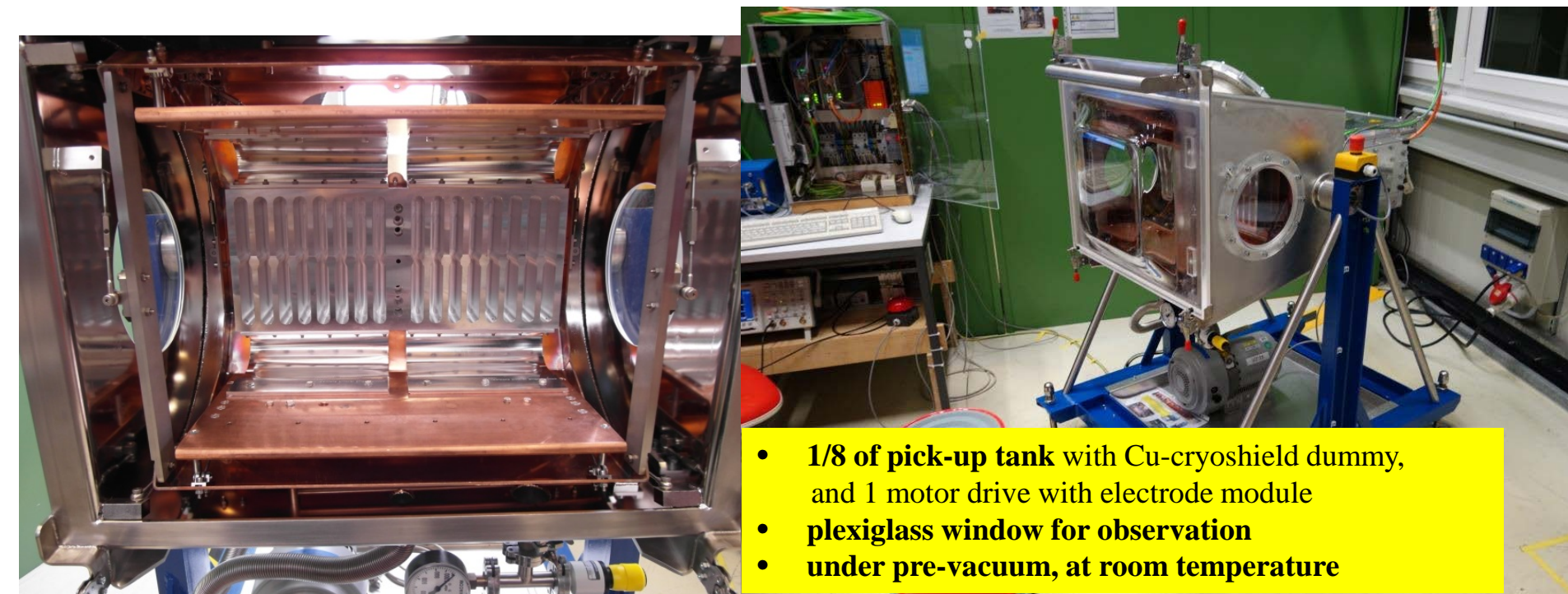


Reached  $\delta p/p$  (rms) =  $2.2 \cdot 10^{-4}$ ,  
 $\varepsilon_x / \varepsilon_y$  (rms) =  $1.9/2.8$  mm mrad in 10 s



Four times (i.e. accounting for statistical beam signal fluctuations) the total max. cw power is just  $\leq$  installed microwave power of 8 kW.

## Linear motor drives for plunging the pick-ups



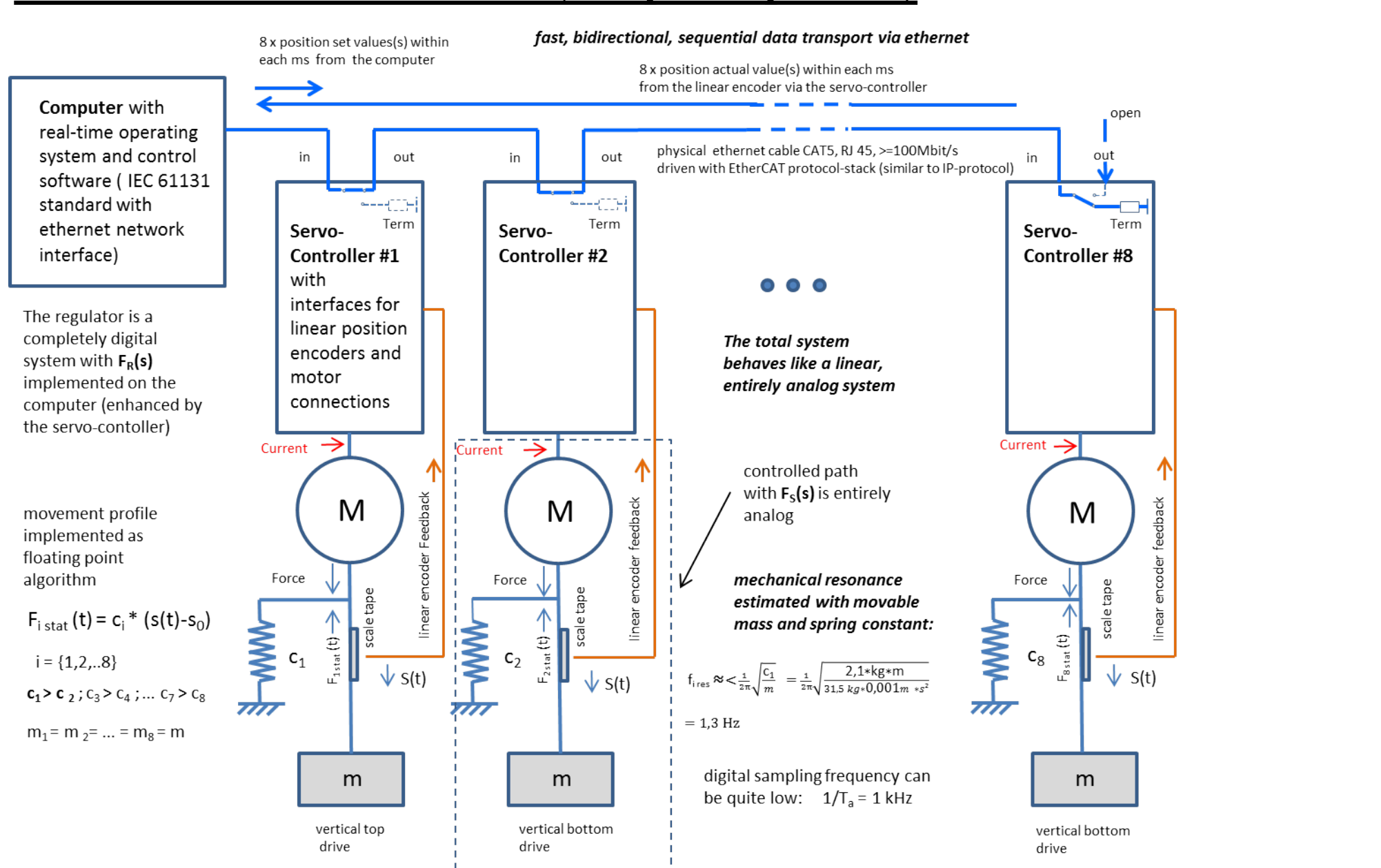
- 1/8 of pick-up tank with Cu-cryoshield dummy, and 1 motor drive with electrode module
- plexiglass window for observation
- under pre-vacuum, at room temperature

Made for mechanical lifetime tests of moving parts, critical measurements of forces/acceleration profiles...

### Features / Requirements

- The load is a resonant system of springs-masses of about 30 kg.
- Stability possible via real-time electronic support/control.
- Motor drives must plunge outwards jerk-free along a path  $S=70$  mm in  $T=0.2$  s.
- In synchronous operation, the middle-point (Schwerpunktabweichung) of 2 motor drives deviates  $< 100 \mu\text{m}$  for horizontal plunging; see more detailed consideration for vertical plunging (with a scale tapes (Maßband) precision of  $5 \mu\text{m}$ )
- Water-cooled motors for CR beam cycles

### Linear drive control scheme (CR pick-up tanks)

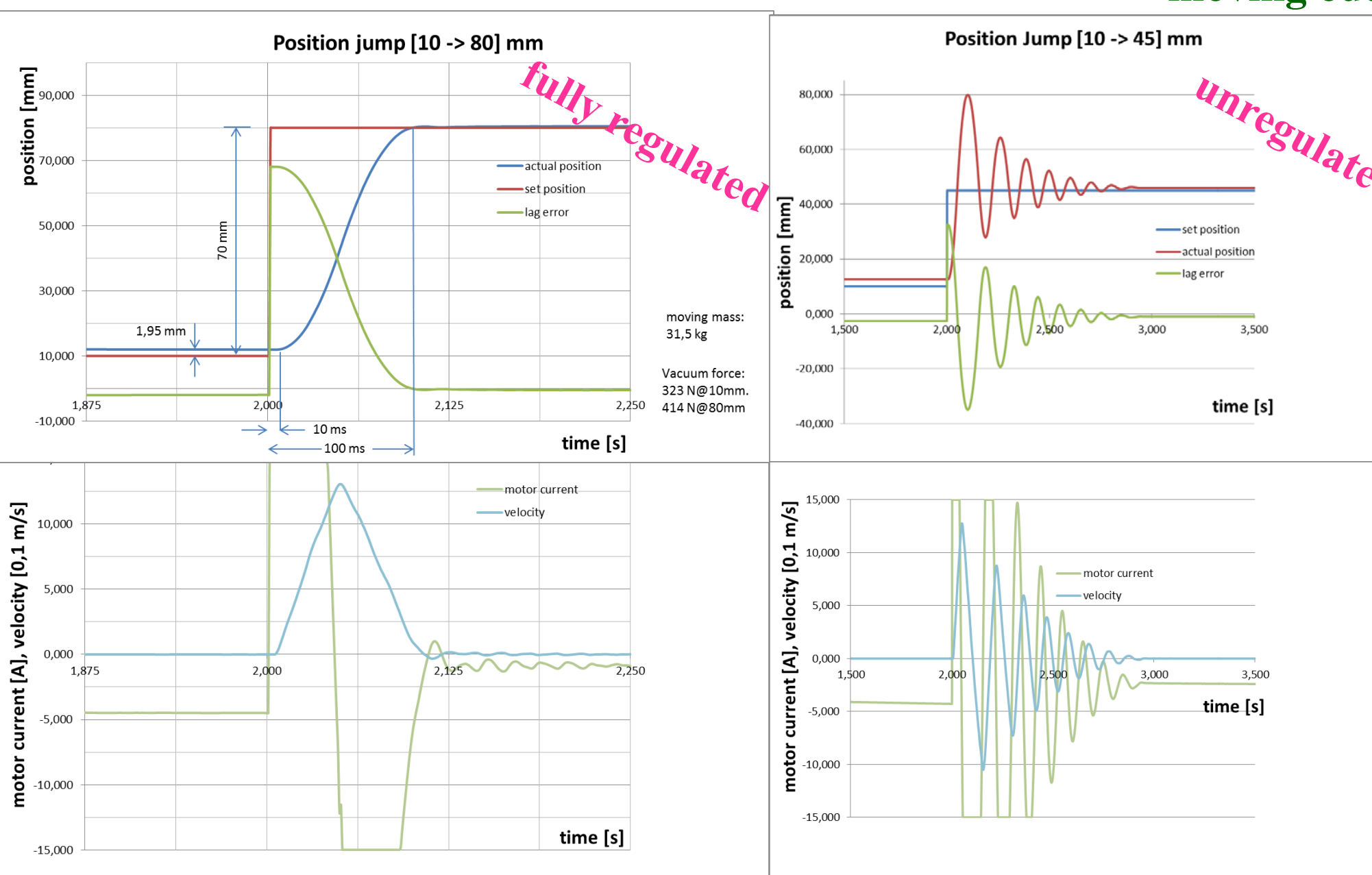


Dynamically most critical case:  
Vertical, top plunging  
Max. positioning velocity at highest static force

### 3 possible orientations

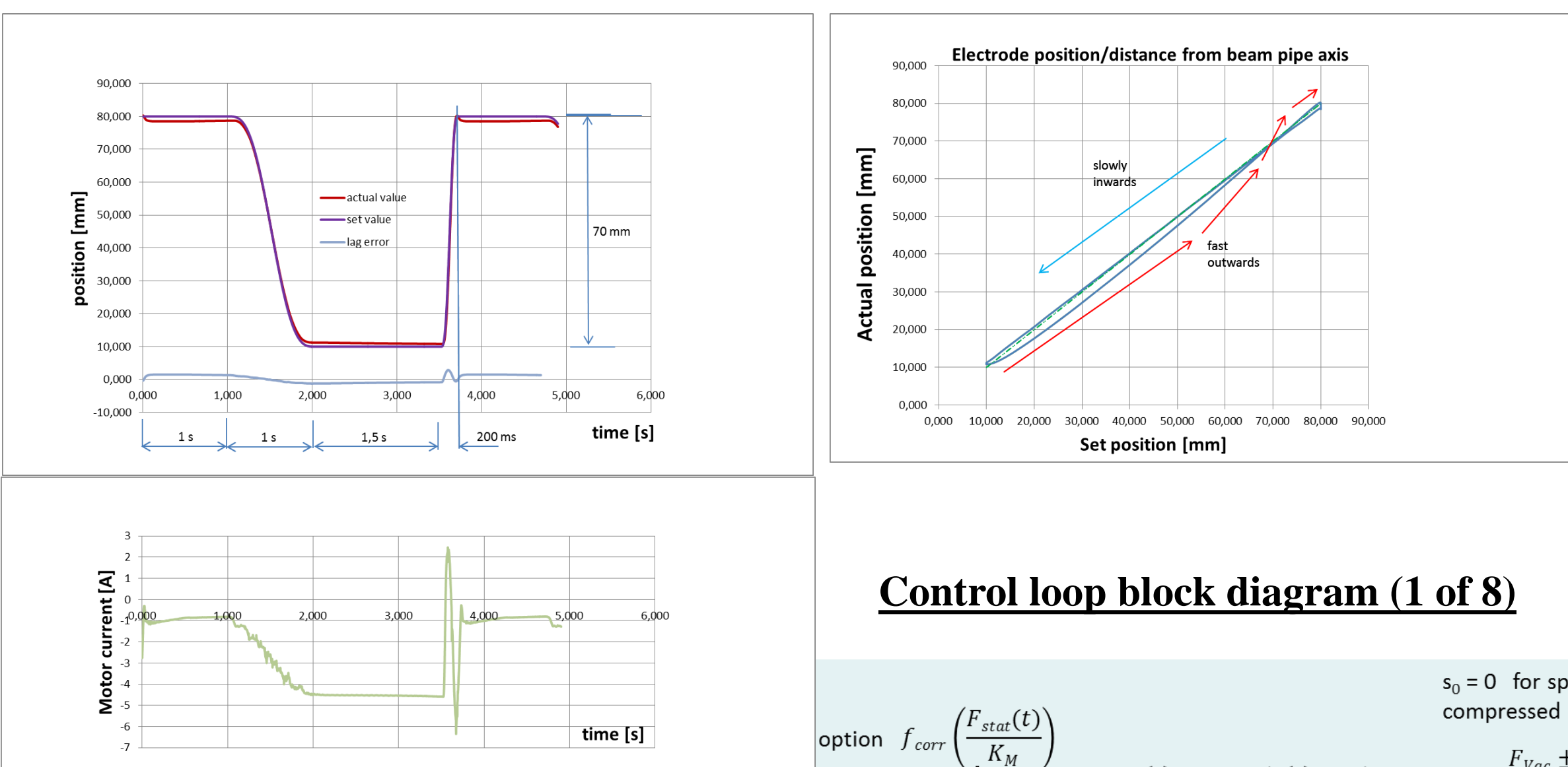
- horizontal
- vertical, motor drive above beam axis, top plunging
- vertical, motor drive below beam axis, bottom plunging

moving outwards from beam axis in max. 120 ms ☺

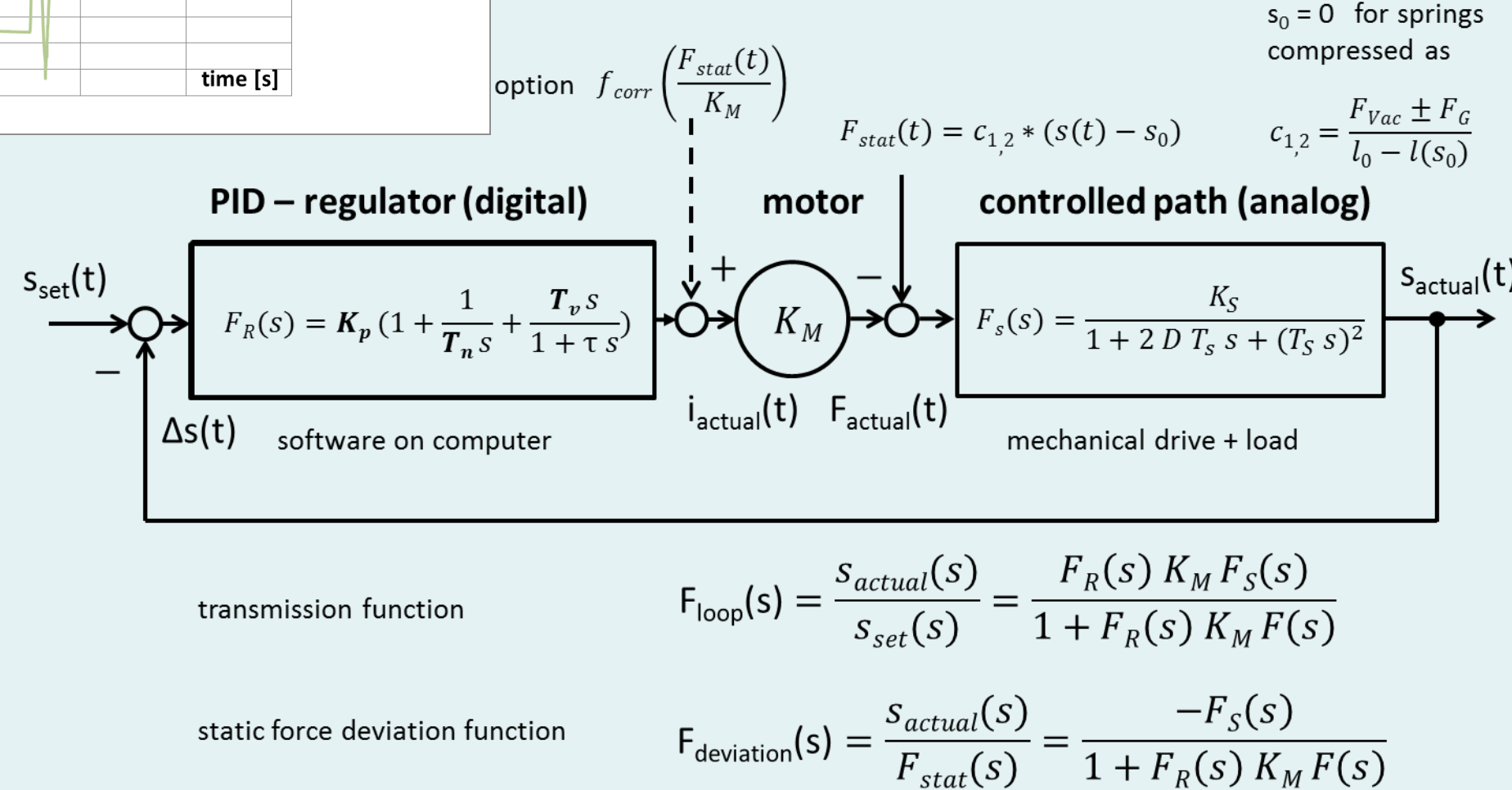


### Closed-loop control characteristics

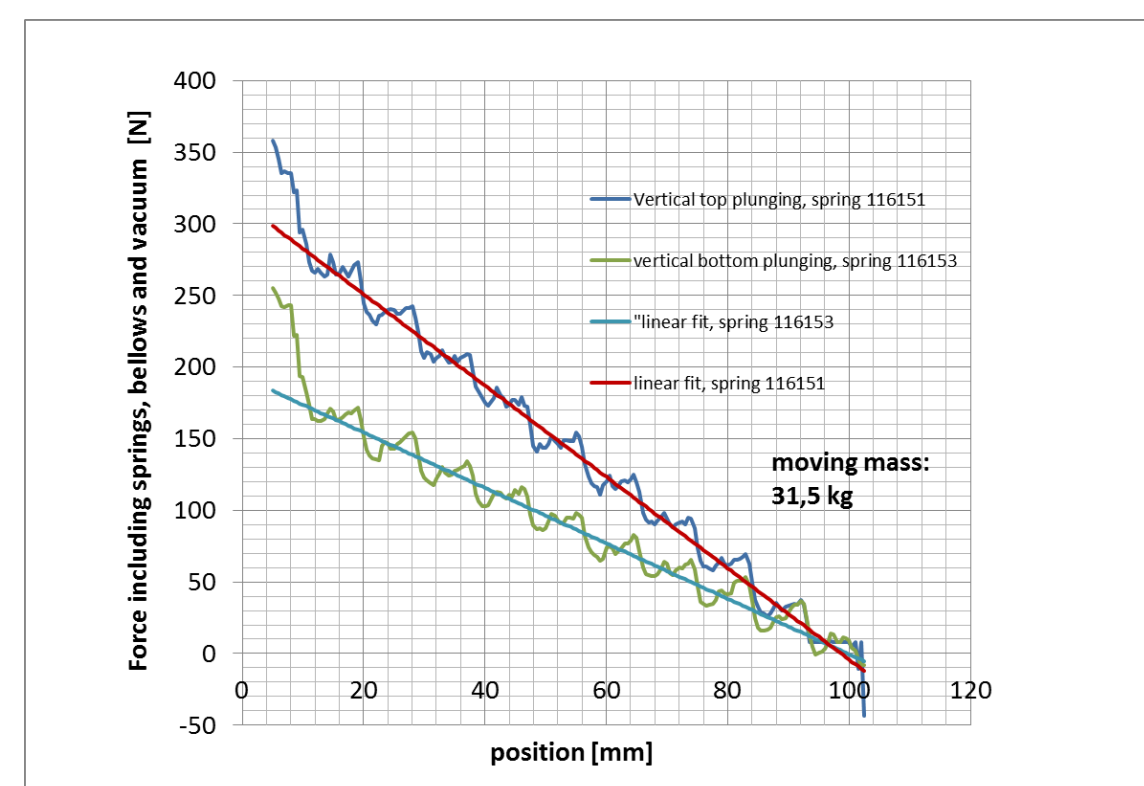
Standard set profile: 1 s inwards / 0,2 s with  $\sin^2$ -shaped velocity outwards



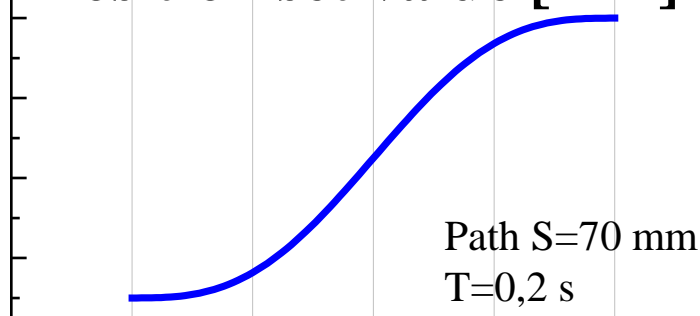
### Control loop block diagram (1 of 8)



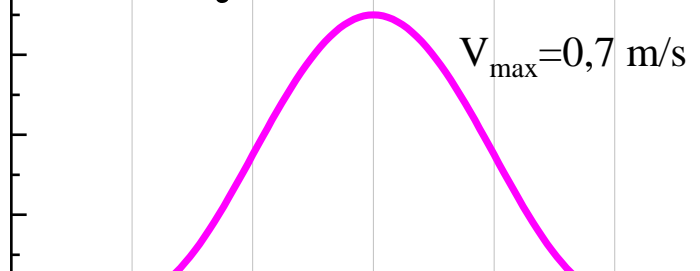
### Static force measurement



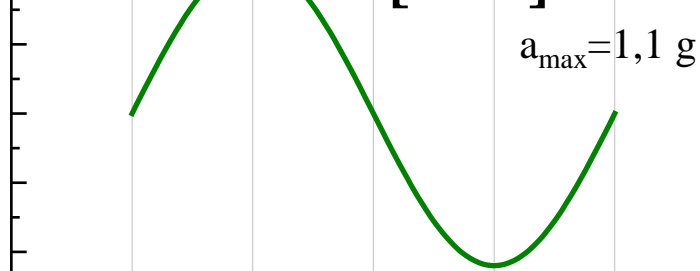
### Position set value [mm]



### Velocity value [m/s]



### Acceleration [m/s^2]



2 linear motor drives tested synchronously inside prototype pick-up tank specification for CR fulfilled!

